

## Sample Probe Design Calculation Datasheet (based on ASME PTC 19.3 TW-2010)

Conax P/N: Z98765-1

Tag Number: SPA/PG5(ASDFG)

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Customer: RAm Systems

**Conax S/O No.:** G99999C

#### Sample Probe Parameters

Probe Material:	316 SST	Fluid Media:	Glycerin
Configuration:		T = Operating Temperature:	100 °F
A = Shank Diameter:	.75 in	P = Operating Pressure:	100 PSI
B = Tip Diameter:		V = Operating Fluid Velocity:	10 ft/s
d = Bore Diameter:	.5 in	V <sub>IR</sub> = In-Line Resonance Velocity:	12.32 m/s
L = Overall Length:	15 in	μ = Dynamic Viscosity:	
U = Unsupported Length (Z+Y):	0.211 m	ν = Kinematic Viscosity:	0.00036 St
Y = Shielding Length	5 in	ρ = Fluid Density	.2 lb/in³
Z = Insertion Length:	3 in	<i>N<sub>Sc</sub></i> = Scruton Number:	0.004
F <sub>i</sub> = Insertion Force:	196.52 N	<i>R<sub>e</sub></i> = Reynolds Number:	2142956.1

Note: The Sample Probe Assembly Configuration falls outside the scope of ASME PTC 19.3 TW-2010, which was specifically developed for Thermowell Configurations. However, based on similar mounting methods and geometries, the analysis methodology of ASME PTC 19.3 TW-2010 will be applied to the Sample Probe Assembly configuration as an effort to validate its design acceptability.

# Acceptance Criteria (Based on values for *R<sub>e</sub>*, *N<sub>Sc</sub>* and Fluid Type, use Acceptance Criteria: Case {x})

- Case 1: For Gaseous Fluid Media and  $N_{sc}$  > 2.5 and  $R_e$  < 10<sup>5</sup>, In-Line Resonance is Suppressed and  $f_s$  < 0.8  $f_n^c$
- Case 2: For Gaseous Fluid Media and if  $N_{Sc}$  > 64 and  $R_e$  < 10<sup>5</sup>, both In-Line and Transverse Resonance is suppressed. Analysis techniques are beyond the scope of ASME PTC 19.3 TW-2010.
- Case 3: For Gaseous Fluid Media and if  $N_{Sc} \le 2.5$  or  $R_e \ge 10^5$ , or for Liquid Media, the following conditions apply: a) If the SPA **PASSES** cyclic stress conditions at In-Line Resonance (S<sub>o,max</sub>), then  $f_s < 0.8 f_n^c$ b) If the SPA **FAILS** cyclic stress conditions at In-Line Resonance (S<sub>o,max</sub>), then  $f_s < 0.4 f_n^c$

S <sub>VM</sub> (Von Mises Stresses) < 1.5 x S <sub>T</sub> (Allowable Material Stress Limit)	1.5 X S <sub>T</sub> =
S <sub>o,max</sub> (Cyclic Stresses) < F <sub>t</sub> F <sub>e</sub> S <sub>f</sub> ((Adjusted Materail Fatigue Limit)	F <sub>e</sub> S <sub>f</sub> = 8,757.7 psi

### Frequency & Stress Analysis Results

Vortex Shedding Frequency (f <sub>s</sub> )	35.2 Hz	$f_r = f_s / f_n^c$ Frequency Ratio	0.165
Resonant Frequency (f <sup>c</sup> )	213.4 Hz	Frequency ratio limit	0.8

Stress Parameter	At Process Velocity (V)		<i>At In-Line Resonance Velocity (V <sub>IR</sub>)</i>	
S <sub>VM</sub> = Von Mises Stresses:	23239 KPa	Pass	371257 KPa	Fail
S <sub>o,max</sub> = Peak Cyclic Stresses:	38542 KPa	Pass	612054133 KPa	Fail

### Conclusion

Frequency Analysis pass/fail: 1 Stress Analysis pass/fail: Fail Sample Probe Assembly design pass/fail: Fail Dampening Factor  $\zeta = 0.0005$